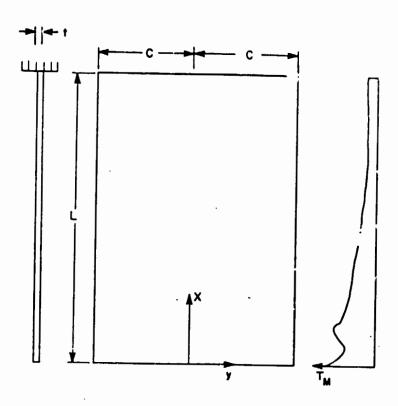
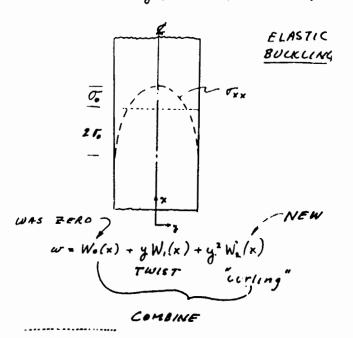
STRESS AND BUCKLING ANALYSIS

UNIVERSITY OF KENTUCKY

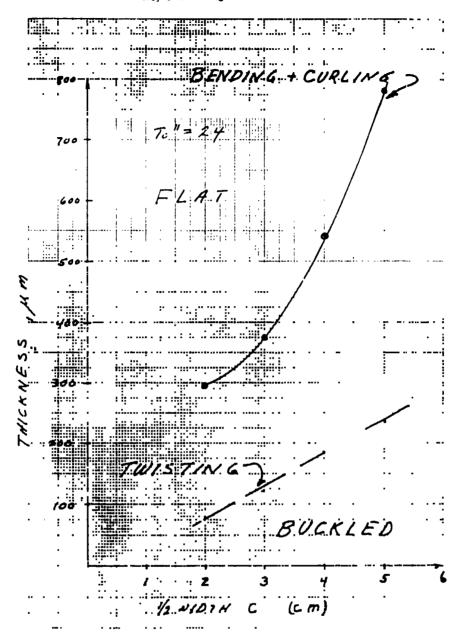
O. Dillon

TECHNOLOGY	REPORT DATE
STRESS AND BUCKLING ANALYSIS	10/3/84 、
APPROACH	STATUS
MODEL MATERIAL BEHAVIOR MODEL BUCKLING DUE TO THERMAL STRESSES	ELASTIC STRESS AND BUCKLING ANALYSIS COMPLETED FOR CONSTANT MATERIAL PRO- PERTIES.
CONTRACTOR UNIVERSITY OF KENTUCKY	CRITICAL SHEET THICKHESS VS. SHEET WIDE COMPARED FOR FOUR THEMAL PRO-
GOALS	FILES.
PROVIDE GUIDANCE BASED ON ANALYSIS FOR THERMAL PROFILES FOR REDUCING STRESSES AND IMPROVING FLATNESS IN WIDE RIBBON.	RESULTS ARE REASONABLY CONSISTENT WITH EXPERIMENT.
HAVE RESULTS BE APPLICABLE TO ALL SHEET GROWTH SYSTEMS.	SURVEY OF THE NECHANICAL PROPERTIES OF SILICON
	PRE-BUCKLED STRESSES IN PLASTIC RANGE

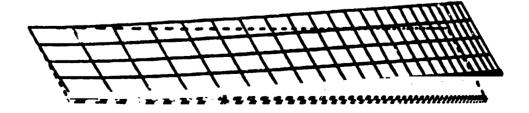




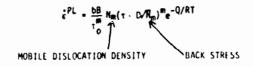
USE ANALYTICAL METHODS TO OBTAIN CRITICAL THICKNESS MINE THE THE STATE OF THE STATE



Calculated Buckled Web Shape

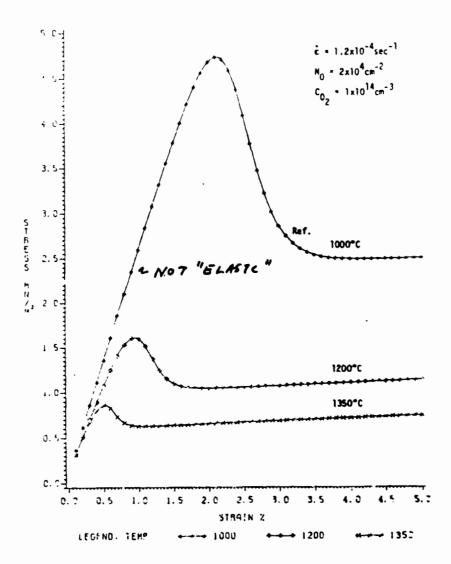


Sumino



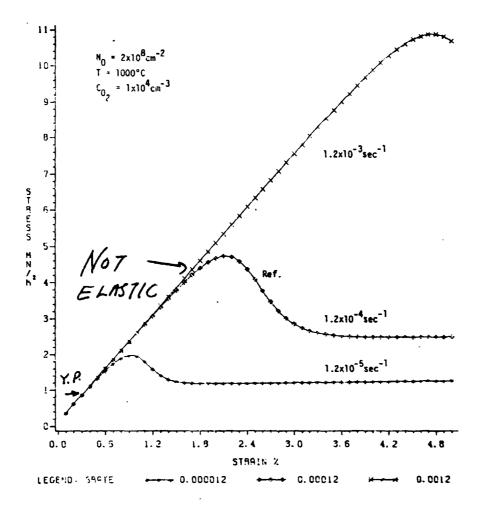
 $N_{r_{c}} = N_{T}N_{m}(1 - D_{c}\Pi_{m})^{A+m}e^{-T/RT}$ DENSITY <u>CHANGES</u>

Stress vs Strain for Si (Te.nperature in °C)

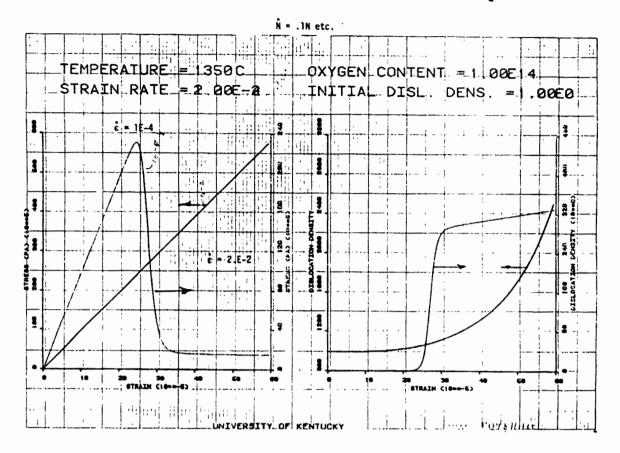


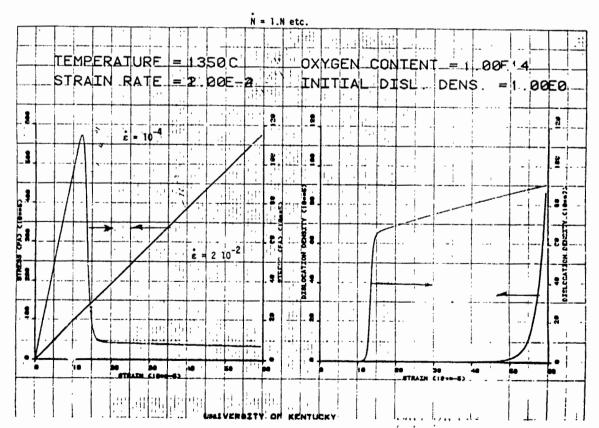
Stress vs Strain for Si (Strain Rate 1/sec)

were seen love to had not it to better



TEMPERATURE=1000 C

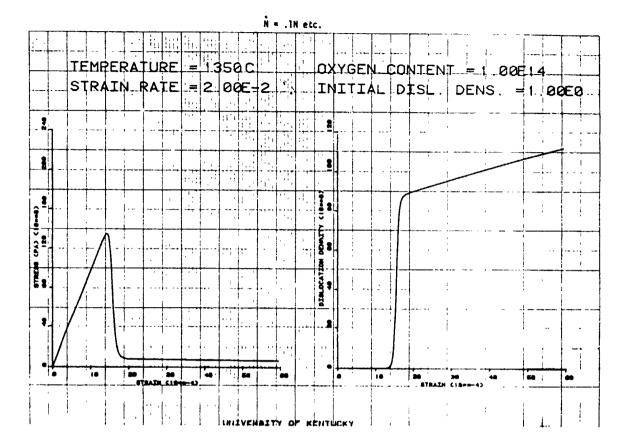




OF POOR QUALITY

The second second

SILICON SHEET



A CHARLEST STATE OF THE STATE OF

Pre-Buckled Stresses

Elastic

APPENDIX A

The equations governing the stress in a ribbon consist of those which reflect that the material is in equilibrium, that the deformations are compatable and that there is a material constitutive relation. Assuming that (a) one can neglect the three stresses z_{1Z} , (b) that the material is elactionard that Young's modulus and Poissons ratio are constant the governing enuatrops are Equilibrium.

$$\frac{3\sigma_{XX}}{3x} + \frac{3\sigma_{XY}}{3y} = 0.$$

$$\frac{3\sigma_{XY}}{3x} + \frac{3\sigma_{YY}}{3y} = 0.$$
(A-1)

Compatability

$$\frac{\partial^2 \varepsilon_{xx}}{\partial y^2} + \frac{\partial^2 \varepsilon_{yy}}{\partial x^2} = 2 \frac{\partial^2 \varepsilon_{xy}}{\partial x \partial y}$$
 (A-2)

Constitutive relations

$$\varepsilon_{xx} = \frac{\sigma_{xx}}{E} - \frac{v}{E}\sigma_{yy} + \alpha T$$
 (A-3)

$$\varepsilon_{yy} = \frac{\sigma_{yy}}{E} - \frac{v}{E} \sigma_{xx} + \alpha T$$

$$\varepsilon_{xy} = \frac{(1+v) \sigma_{xy}}{E}$$
(A-4)

Equations (A-1) combine to yield

$$\frac{\partial^2 \sigma_{xx}}{\partial x^2} = \frac{\partial^2 \sigma_{yy}}{\partial y^2} \qquad (\cdots,$$

Thile Eqs. (A-2) through (A-4) yield

$$\frac{\partial^2 \sigma_{yy}}{\partial x^2} = -\frac{\partial^2 \sigma_{xx}}{\partial x^2} - \frac{\partial^2 \sigma_{yy}}{\partial y^2} - \frac{\partial^2 \sigma_{xx}}{\partial y^2} - \frac{\partial^2 \sigma_{xx}}{\partial x^2} - \frac{\partial^2 \sigma^2}{\partial x^2}$$
 (A-c)

Second order central differences equivalents of these equations are:

$$\sigma_{XX}(I+1,J) = 2\sigma_{XX}(I,J) - \sigma_{XX}(I-1,J)$$

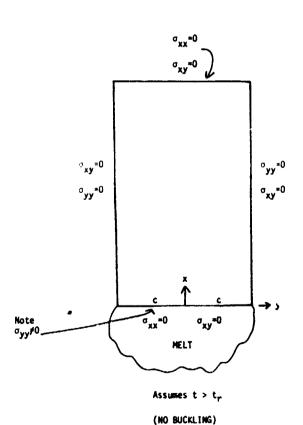
$$+ [\sigma_{yy}(I,J+1) - 2\sigma_{yy}(I,J)$$

$$+ \sigma_{yy}(I,J-1)] \cdot x^{2}/2y^{2}$$
1-5a

and

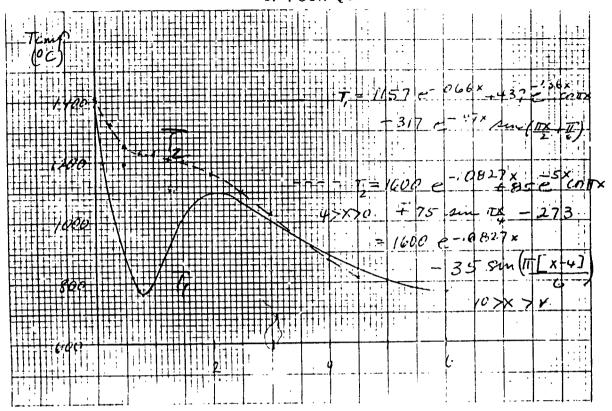
and the state of t

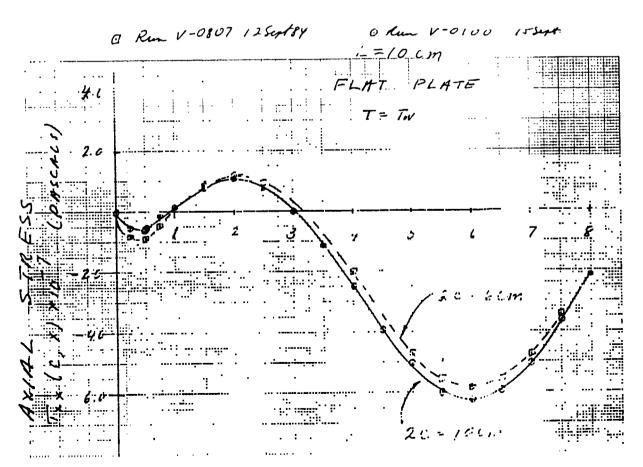
$$\begin{split} \sigma_{yy}(I+1,J) &= 2\sigma_{xx}(I,J) - \sigma_{yy}(I-1,J) \\ &- \sigma_{xx}(I+1,J) + 2\sigma_{xx}(I,J) \\ &- \sigma_{yx}(I-1,J) - \frac{\alpha E \sigma^2 T}{\sigma^2 x^2} \Delta x^2 \\ &- (\frac{\Delta x}{\Delta y})^2 \left[\sigma_{xx}(I,J+1) - 2\sigma_{xx}(I,J) \right. \\ &+ \sigma_{xx}(I,J-1) + \sigma_{yy}(I,J+1) - 2\sigma_{yy}(I,J) \\ &- \\ &+ \sigma_{yy}(I,J-1) \right] \end{split}$$
(A-6a)



market to be a facility of the second

ORIGINAL PRODUCT





He assume the Sumino model of viscoplastic behavior, i.e.

$$\tilde{\epsilon}_{ij}^{PL} = \frac{bB}{\tau_0^m} \frac{N_m (/J_2 - D/N_m)^m e^{-Q/kT} (\sigma_{ij} - \sigma_{KK} \delta_{ij}/3)}{/J_2}$$

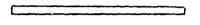
DISLOCATION DENSITY CHANGES!!

$$\hat{\mathbf{N}}_{m} = \mathbf{K}_{1} \ \mathbf{N}_{m} (\tau - \mathbf{D} / \tilde{\mathbf{N}}_{m})^{\lambda + m} \mathrm{e}^{-Q/k_{1}}$$

MOBILE

NEW:

SOME BASIS FOR A "SHAPE" FACTOR



VS.



A
b
B
To
MATERIAL "CONSTANTS"
C
K1
Q



$$\frac{a^2 \dot{\sigma}_{xx}}{ax^2} = \frac{a^2 \dot{\sigma}_{yy}}{ay^2} \tag{1}$$

The compatibility equation is

$$\frac{\partial^2 \dot{\varepsilon}_{xx}}{\partial y^2} + \frac{\partial^2 \dot{\varepsilon}_{yy}}{\partial x^2} = 2 \frac{\partial^2 \dot{\varepsilon}_{xy}}{\partial x \partial y}$$
 (2)

$$\dot{\hat{\epsilon}}_{ij} = \frac{1+\omega}{E} \dot{\hat{\sigma}}_{ij} - \frac{\nu}{E} \dot{\hat{\sigma}}_{KK} \delta_{ij} + \omega \dagger \delta_{ij} + \dot{\hat{\epsilon}}_{ij}^{PL}$$
(3)

Eqs (2) and (3), to yield

$$\frac{\partial^{2}(\dot{\sigma}_{xx} + \dot{\sigma}_{yy})}{\partial x^{2}} + \frac{\partial^{2}}{\partial y^{2}}(\dot{\sigma}_{xx} + \dot{\sigma}_{yy})$$

$$= - E\alpha(\frac{\partial^{2}}{\partial x^{2}} + \frac{\partial^{2}}{\partial y^{2}}) \dagger$$

$$+ (\frac{\partial^{2} \dot{\epsilon}_{xx}}{\partial y^{2}} + \frac{\partial^{2} \dot{\epsilon}_{yy}}{\partial x^{2}} - 2\frac{\partial^{2} \dot{\epsilon}_{xy}}{\partial x \partial y}) E$$
(4)

ORIGINAL PRODUCT SILICON SHEET

PLASTIC

the work with the second

$$\frac{3^2(\sigma_{XX}+\sigma_{YY})}{3^2c^2PL} + \frac{3^2(\sigma_{XX}+\sigma_{YY})}{3^2c^2PL} = -\alpha E \frac{3^2T}{3^2C^2PL}$$

+
$$\int_{0}^{x} \frac{E}{Y} \left(\frac{3^{2} e^{PL}}{3y^{2}} + \frac{3^{2} e^{PL}}{3x^{2}} - 2 \frac{3^{2} e^{PL}}{3x3y} \right) dx$$
Pull RATE

$$\frac{\partial^2 \sigma_{xx}}{\partial x^2} = \frac{\partial^2 \sigma_{yy}}{\partial y^2}$$

 $\dot{\varepsilon}_{ij}^{PL}$ = function of stresses and N $_{\rm m}$

$$\hat{\mathbf{H}}_{m} = \mathbf{K}_{1} \mathbf{N}_{m} (\tau - \mathbf{D} / \hat{\mathbf{N}}_{m})^{\lambda + m} e^{-\mathbf{Q} / RT}$$

SOLVE VIA INTERATION!!!!

"0" = ELASTIC

20

OUTPUT STRESSES (x,y)

STRAIN RATES (x,y)

VERY NEW -> DISLOCATION DENSITY (x.y)

L = 1 cm
$$\stackrel{\circ}{\epsilon} = 10^{-2} \text{ sec}^{-1}$$

C = 2 cm
T = Tw $\stackrel{\circ}{\epsilon}^{PL} = 10^{-7} \text{ sec}^{-1}$

$$o_{xx_{max}}^{PL} = .1147 \times 10^{8} \text{ pascals}$$
 $o_{xx_{max}}^{PL} = .1086 \times 10^{8} \text{ pascals}$

$$N_{init} = 1.0/cm^2$$

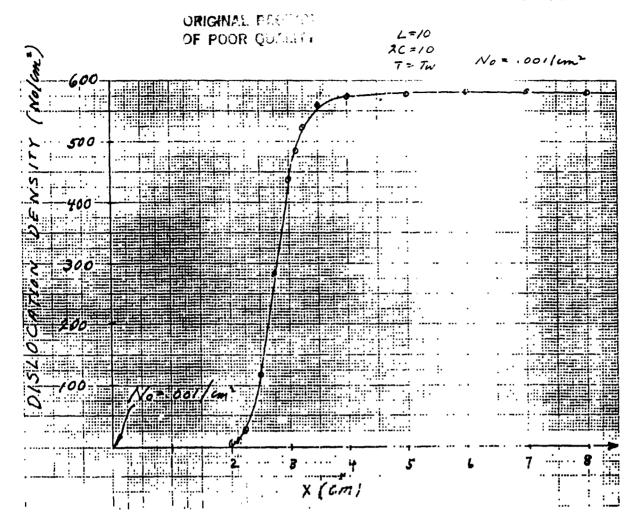
$$N_{final} = 12/cm^2$$
 at outer edge $6/cm^2$ at center

EVERY PROBLEM DIFFERENT

$$\dot{c} = 10^{-3} \rightarrow 10^{-2} \text{ sec}^{-1}$$

$$\dot{c}^{PL} = 10^{-9} + 10^{-4} \text{ sec}^{-1}$$

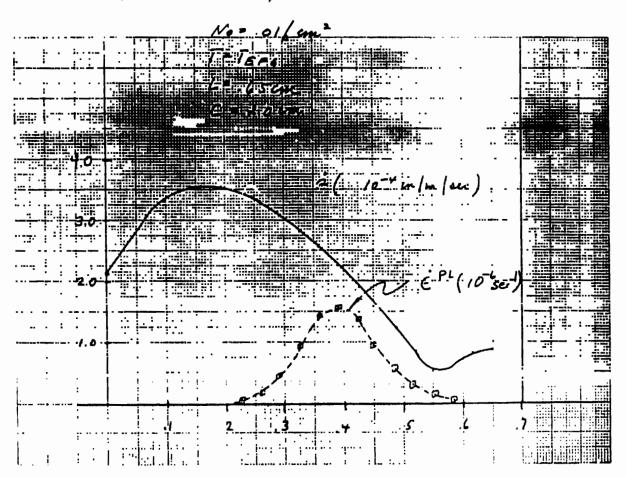
$$+ 10^{-3} \text{ sec}^{-1} \quad \text{once in awhile}$$



rolan?	Teris	- 1 1 	
2 600	No = .00; /cm		
ENS 1	L = .65cm.		
400			
0 300	<i></i>		
0 200 0		NOTE:	
57 100	6= 001/om2 3	•	10 50 cm
	.1 .2 .3 .4	.5 -	
	X (cm;		

ORIGINAL PAGE IS OF POOR QUALITY

Run V-0891 2184+



NUMERICAL PROCEDURE CONVERGES RAPIOLY change in o is = 10⁻⁴o

DOES NOT CONVERGE

NOTE 1: ELASTIC REGION NO PROBLEM

NOTE 2: DUES NOT CONVERGE

MEANS
$$\epsilon^{PL} * 10^3 \text{ sec}^{-1}$$

$$N_m = 10^{50}/cm^2$$

MOTE 3:

MAJOR OBSERVATION

L = .65 cm CONVERCE

L = .70 cm COES NOT

L = 0.5 + 10. CONVERGES WITH Earl TEMP PROFILE

FOR STRESSES USE ELASTIC

FOR RESIDUAL STRESSES

PLASTICITY

chij(x,y)

PLASTICITY IS SMALL BUT IS JUST AS BAD AS A LOT.

έ is HIGH σ_{1j} APE HIGH

T = 750° x + .75 cm

MODEL INADEQUATE

Re: DEFORMATION MODE CHANGES (twins)

Problems and Concerns

- 1. No at MELT INTERFACE (x=0)
- 2. VALIDITY OF CONSTRANTS FOR SUMING MODEL FOR RIPBON
- 3. SHOULD WE THINK "TWINS"?
- 4. SHOULD WE WORK OTHER PROFILES?
- 5. ROLE OF CHANGING "L"